AMENDMENTS TO THE SPECIFICATION

Please replace Paragraph [0040] with the following paragraph rewritten in amendment format:

[0040] Referring now to Figure 2. Although the details of construction of floating seal 82 are not part of the present invention, for exemplary purposes seal 82 is of a coaxial sandwiched construction and comprises an annular base plate 100 having a plurality of equally spaced upstanding integral projections 102. Disposed on plate 100 is an annular gasket 106 having a plurality of equally spaced holes which receive projections 102. On top of gasket 106 is disposed an upper seal plate 110 having a plurality of equally spaced holes which receive ~~base portions 104~~ projections 102. Seal plate 110 has disposed about the inner periphery thereof an upwardly projecting planar sealing lip 116. The assembly is secured together by swaging the ends of each of the projections 102, as indicated at 118.

Please replace Paragraph [0046] with the following paragraph rewritten in amendment format:

[0046] As best seen with reference to Figures 2 and 5 through 7, valving ring 150 comprises a generally circular shaped main body portion 156 having a pair of substantially diametrically opposed radially inwardly extending protrusions 158 and 160 provided thereon of substantially identical predetermined axial and circumferential dimensions. Suitable substantially identical circumferentially extending guide surfaces 162, 164 and 166, 168 are provided adjacent axially opposite sides of protrusions 158 and 160, respectively. Additionally, two pairs of substantially identical circumferentially

extending axially spaced guide surfaces 170, 172 an 174, 176 are provided on main body 156 being positioned in substantially diametrically opposed relationship to each other and spaced circumferentially approximately 90° from respective protrusions 158 and 160. As shown, guide surfaces 172 and 174 project radially inwardly slightly more from main body 156 as do guide surfaces 162 and 166. Preferably, guide surfaces 172, 174 and 162, 166 are all axially aligned and lie along the periphery of a circle of a radius slightly less than the radius of main body 156. Similarly, guide surfaces 170 and 176 project radially inwardly slightly more from main body 156 as do guide surfaces 164 and 168 with which they are preferably axially aligned. Also surfaces 170, 176 and 164, 168 lie along the periphery of a circle [[or]] <u>of</u> a radius slightly less than the radius of main body 156 and preferably substantially equal to the radius of the circle along which surfaces 172, 174 and 162, 166 lie. Main body [[158]] <u>156</u> also includes a circumferentially extending stepped portion 178 which includes an axially extending circumferentially facing stop surface 180 at one end. Step portion 178 is positioned between protrusion 160 and guide surfaces 170, 172. A pin member 182 is also provided extending axially upwardly adjacent one end of stepped portion 178. Valving ring 150 may be fabricated from a suitable metal such as aluminum or alternatively may be formed from a suitable polymeric composition and pin 182 may be either pressed into a suitable opening provided therein or integrally formed therewith.

Please replace Paragraph [0048] with the following paragraph rewritten in amendment format:

[0048] Groove 186 is sized to movably accommodate protrusions 158 and

160 when valving ring is assembled thereto and notches 188 and 190 are sized to enable protrusions 158 and 160 to be moved into groove 186. Additionally, cylindrical portion 184 will have a diameter such that guide surfaces 162, 164, 166, 168, 170, 172, 174 and [[76]] 176 will slidingly support rotary movement of valving ring 150 with respect to non-orbiting scroll member 66.

Please replace Paragraph [0050] with the following paragraph rewritten in amendment format:

[0050] As best seen with reference to Figure 9, actuating assembly 152 includes a piston and cylinder assembly 200 and a return spring assembly 202. Piston and cylinder assembly 200 includes a housing 204 having a bore defining a cylinder 206 extending inwardly from one end thereof and within which a piston 208 is movably disposed. An outer end 210 of piston 208 projects axially outwardly from one end of housing 204 and includes an elongated or oval-shaped opening 212 therein adapted to receive pin 182 forming a part of valving ring 150. Elongated or oval opening 212 is designed to accommodate the arcuate movement of pin 182 relative to the linear movement of piston end 210 during operation. A depending portion 214 of housing 204 has secured thereto a suitably sized mounting flange 216 which is adapted to enable housing 204 to be secured to a suitable flange member 218 by bolts 220. Flange 218 is in turn suitably supported within outer shell 12 such as by bearing housing [[18]] 20.

Please replace Paragraph [0062] with the following paragraph rewritten in amendment format:

[0062] When the load conditions change to the point that the full capacity of compressor 10 is not required, sensors 296 will provide a signal indicate thereof to controller 298 which in turn will deenergize coil 284 of solenoid assembly 268. Plunger 276 will then move outwardly from tubular member 270 under the biasing action of spring [[276]] 278 thereby moving valve member [[278]] 280 into sealing engagement with seat [[280]] 282 thus closing off passage 256 and the flow of pressurized fluid therethrough. It is noted that recessed area 248 will be in continuous fluid communication with open recess 72 and hence continuously subject to discharge pressure. This discharge pressure will aid in biasing valve member 280 into fluid tight sealing engagement with valve seat 282 as well as retaining same in such relationship.

Please replace Paragraph [0065] with the following paragraph rewritten in amendment format:

[0065] While the above embodiment has been described utilizing a passage 228 provided in housing 204 to vent actuating pressure from cylinder 206 to thereby enable compressor 10 to return to reduced capacity, it is also possible to delete passage 228 and incorporate a vent passage in the valve body 236 in place thereof. Such an embodiment is shown in Figures 13 and 14. Figure 13 shows a modified valve body 236' incorporating a vent passage 312 which will operate to continuously vent passage [[252"]] 252 to suction pressure and hence allow cylinder 206 to vent to suction via line 266. Figure 14 in turn shows a modified piston and cylinder assembly 200' in which vent passage 228 has been deleted. The operation and function of valve body 236' and piston cylinder assembly 200' will otherwise be substantially identical to that

disclosed above. Accordingly, corresponding portions of valve bodies 236 and 236', piston and cylinder assemblies 200 and 200' are substantially identical and have each been indicated by the same reference numbers.

Please replace Paragraph [0066] with the following paragraph rewritten in amendment format:

[0066] While the above embodiments provide efficient relatively low cost arrangements for capacity modulation, it is also possible to utilize a three way solenoid valve in which the venting of cylinder [[230]] 206 is also controlled by valving. Such an arrangement is illustrated and will be described with reference to Figure 15. In this embodiment, a valve body 314 is secured to shell 12 in the same manner as described above and includes an elongated central bore 316 within which is movably disposed a spool valve 318. Spool valve 318 extends outwardly through shell 12 into solenoid coil 320 and is adapted to be moved longitudinally outwardly from valve body 314 upon energization of solenoid coil 320. A coil spring 322 operates to bias spool valve 318 into valve body 314 when coil 320 is not energized.

Please replace Paragraph [0075] with the following paragraph rewritten in amendment format:

[0075] Non-orbiting scroll member 466 is also provided having wrap 64 positioned in meshing engagement with wrap 54 of orbiting scroll 50. Non-orbiting scroll 466 has a centrally disposed discharge passage which communicates with upwardly open recess 72 which in turn is in fluid communication via opening 74 in partition 18

with discharge muffler chamber 76 defined by end cap 14 and partition 18. Non-orbiting scroll member 466 has in the upper surface [[there]] thereof annular recess 80 having parallel coaxial sidewalls in which is sealingly disposed for relative axial movement annular floating seal 82 which serves to isolate the bottom of recess 80 from the presence of gas under suction pressure and gas under discharge pressure so that it can be placed in fluid communication with a source of gas at an intermediate fluid pressure by means of passageway 84. Non-orbiting scroll member 466 is thus axially biased against orbiting scroll member 50 to enhance wrap tip sealing by the forces created by discharge pressure acting on the central portion of scroll member 466 and those created by intermediate fluid pressure acting on the bottom of recess 80. Discharge gas is also sealed from gas at suction pressure in shell 12 by means of a seal acting against annular wear ring 132 attached to partition 18. Non-orbiting scroll member 466 is designed to be mounted to bearing housing 20 in a suitable manner which will provide limited axial (and no rotational) movement of non-orbiting scroll member 466.

Please replace Paragraph [0083] with the following paragraph rewritten in amendment format:

[0083] Referring now to Figure 18, a compressor 410' is shown. Compressor 410' is the same as compressor 410 except that solenoid valve 412 has been replaced by solenoid valve 412'. Solenoid valve 412' is located outside of shell 12 as opposed to solenoid valve 412 which is located within shell 12. A fluid pipe 422 extends through a fitting 424 attached to shell 12 to place solenoid valve 412' in communication with recess 80. A fluid pipe 426 extends between solenoid valve 412'

and suction inlet fitting 22 to place solenoid valve 412' in communication with the suction pressure zone of compressor [[410]] 410'. The function and operation of compressor 410' and solenoid valve 412' are the same as described above for compressor 410 and solenoid valve 412.

Please replace Paragraph [0096] with the following paragraph rewritten in amendment format:

[0096] Referring now to Figure 22, a compressor 510' is shown. Compressor 510' is the same as compressor 510 except that solenoid valve 532 has been replaced by solenoid valve 532'. Solenoid valve 532' is located outside of shell 12 as opposed to solenoid valve 532 which is located within shell 12. A fluid pipe 542 extends through a fitting 544 attached to end cap 514 to place solenoid valve 532' in communication with annular chamber 580. A fluid pipe 546 extends between solenoid valve 532' and suction inlet fitting 522 or is otherwise connected to the suction chamber of compressor [[510]] 510' to place solenoid valve 532' in communication with the suction pressure zone of compressor 510. The function and operation of compressor 510' and solenoid valve 532' are the same as described above for compressor 510 and solenoid valve 532.